

Solutions to Homework 5Question 1: 10 points

Problem Statement. Figure 1 shows a simple three-bar truss. The bar elements have section properties EA throughout. Horizontal and vertical loads P are applied at node C.

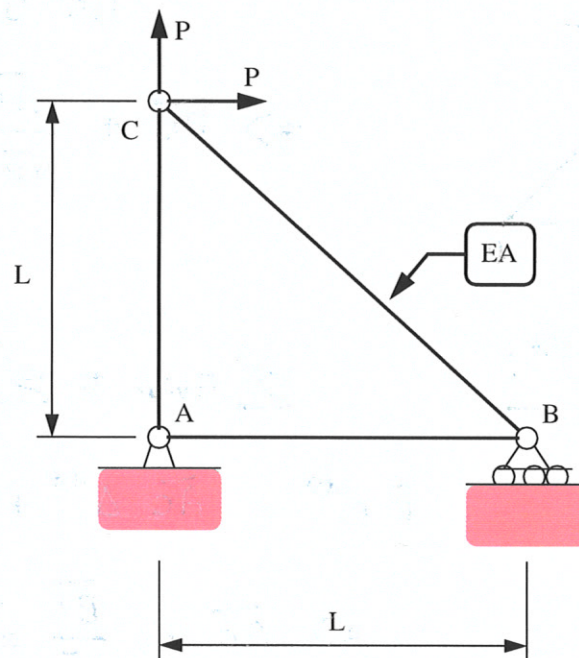


Figure 1: Simple three-bar truss.

Part [1a]. (5 pts). Use the **method of virtual displacements** to compute the vertical reaction forces at nodes A and B. Show all of your working.

Handwritten solution for Part [1a]:

Virtual displacement diagrams for nodes A and B are shown. For node A, a vertical virtual displacement Δ_A^{**} is indicated. For node B, a horizontal virtual displacement Δ_B^{**} is indicated.

Virtual work equations:

$$\sum \text{EWD} = 0 \quad V_A \Delta_A^{**} + \Delta_A^{**} P + \Delta_A^{**} P = 0$$

$$\Rightarrow V_A = -2P$$

For node B:

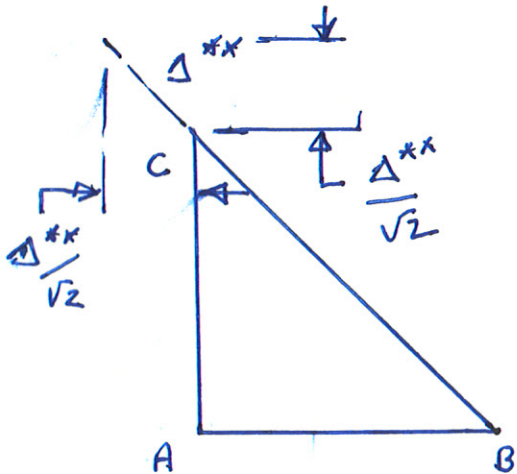
$$\sum \text{EWD} = 0$$

$$V_B \Delta_B^{**} + P(-\Delta_B^{**}) = 0$$

$$\Rightarrow V_B = P$$

Part [1b]. (5 pts). Use the **method of virtual displacements** to compute the member forces AC and BC. Show all of your working.

Extend BC by Δ^{**} :

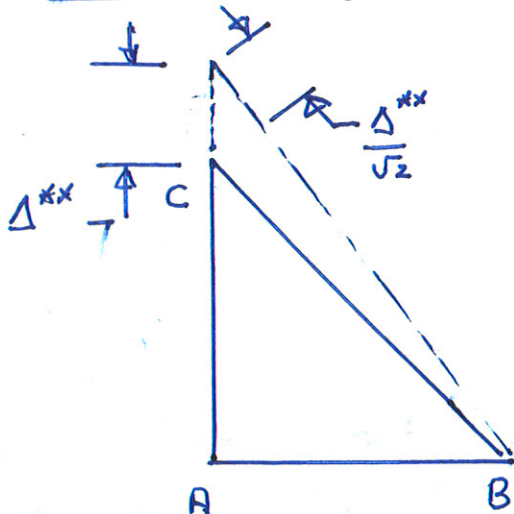


$$\sum EWD = \sum IWD.$$

$$\left(\frac{\Delta^{**}}{\sqrt{2}}\right)(-P) + \left(\frac{\Delta^{**}}{\sqrt{2}}\right)P = \overline{BC} \Delta^{**} + \overline{AC} \left(\frac{\Delta^{**}}{\sqrt{2}}\right)$$

$$\Rightarrow \overline{BC} + \frac{\overline{AC}}{\sqrt{2}} = 0 \quad \text{--- (A)}$$

Extend AC by Δ^{**} :



$$\sum EWD = \sum IWD.$$

$$P \Delta^{**} = \overline{AC} \Delta^{**} + \overline{BC} \left(\frac{\Delta^{**}}{\sqrt{2}}\right)$$

$$\Rightarrow \overline{AC} + \frac{\overline{BC}}{\sqrt{2}} = P. \quad \text{--- (B)}$$

Combining equations (A) & (B)

$$\overline{AC} = 2P$$

$$\overline{BC} = -\sqrt{2}P.$$

Question 2: 10 points

Problem Statement. The cantilevered beam structure shown in Figure 2 supports a uniformly distributed load w (N/m) between points C and D.

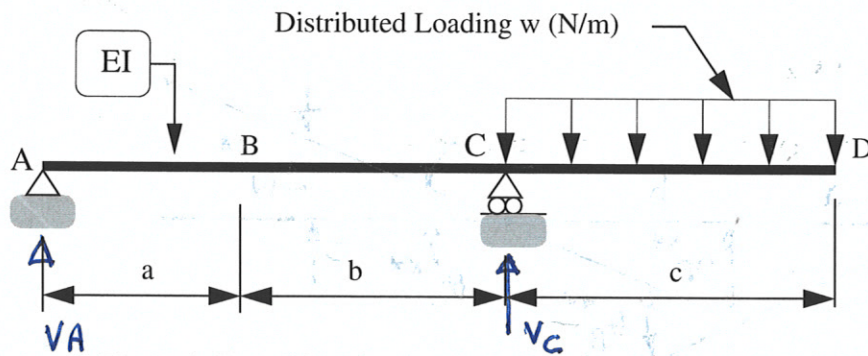
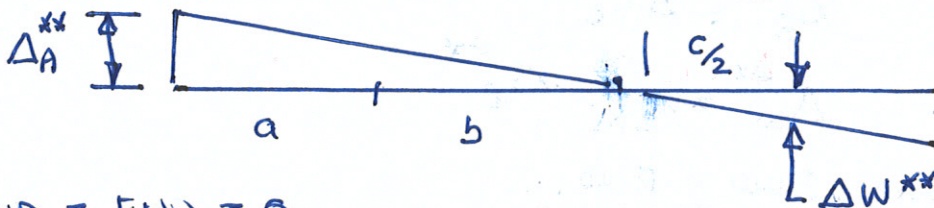


Figure 2: Front elevation view of a simple beam structure.

Part [2a]. (4 pts) Use the method of **virtual displacements** to compute formulae for the vertical reactions at A and C. Show all of your working.

Virtual Displacement at A.



From geometry:

$$\Delta_W^{**} = \frac{c}{2(a+b)} \cdot \Delta_A^{**}$$

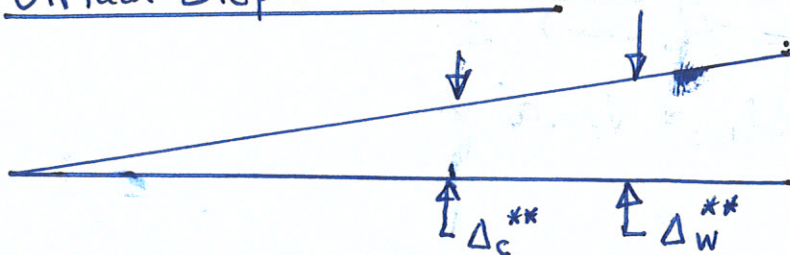
$$\text{IWD} = \text{END} = 0$$

$$\text{END} = V_A \cdot \Delta_A^{**} + \frac{w c^2}{2(a+b)} \Delta_A^{**} = 0$$

$$\Rightarrow V_A = -\frac{w c^2}{2(a+b)}$$

(A)

Virtual Displacement at C.



From geometry:

$$\Delta_W^{**} = \frac{a+b+c/2}{(a+b)} \Delta_C^{**}$$

$$\text{END} = 0$$

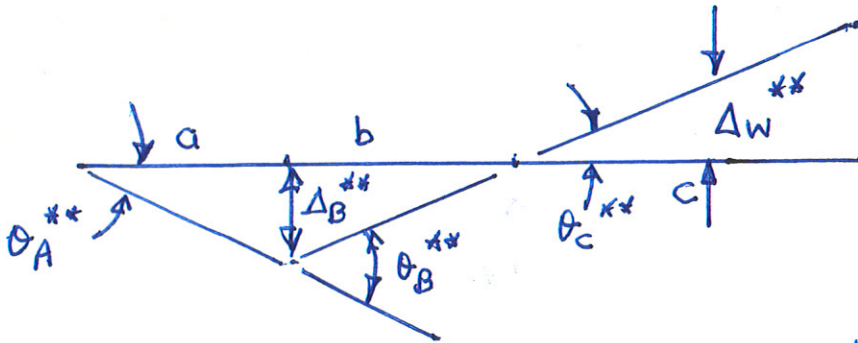
$$\Rightarrow \Delta_C^{**} V_C - w \Delta_C^{**} \frac{a+b+c/2}{(a+b)} = 0$$

$$\Rightarrow V_C = \frac{w(a+b+c/2)}{(a+b)}$$

(B)

Part [2b]. (6 pts) Use the method of **virtual displacements** to compute a formula for the bending moment at B. Show all of your working.

Apply virtual rotation at B.



From geometry: $\theta_B^{**} = \theta_A^{**} + \theta_C^{**} = \frac{\Delta_B^{**}}{a} + \frac{\Delta_B^{**}}{b}$

$$\Delta_B^{**} = a \theta_A^{**}$$

$$\Delta_B^{**} = b \theta_C^{**}$$

$$\Delta_w^{**} = \frac{c}{2b} \cdot \Delta_B^{**}$$

Energy Balance : $IWD = EWD$.

$$M_B \theta_B^{**} + w_c \Delta_w^{**} = 0$$

$$\Rightarrow M_B^{**} \left[\frac{1}{a} + \frac{1}{b} \right] \Delta_B^{**} = \frac{-w_c^2}{2b} \Delta_B^{**}$$

$$\Rightarrow M_B = \left(\frac{ab}{a+b} \right) \left[\frac{-w_c^2}{(2b)} \right]. \checkmark$$

Note: From statics: $M_B = V_A \cdot a \checkmark \checkmark$.

Question 3: 10 points

Problem Statement. Consider the articulated cantilever beam structure shown in Figure 3.

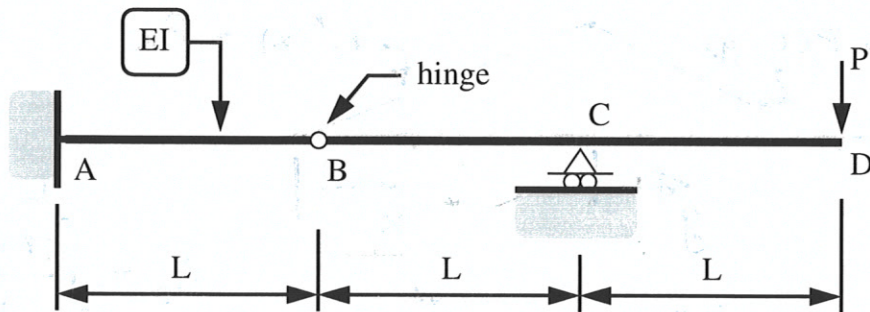
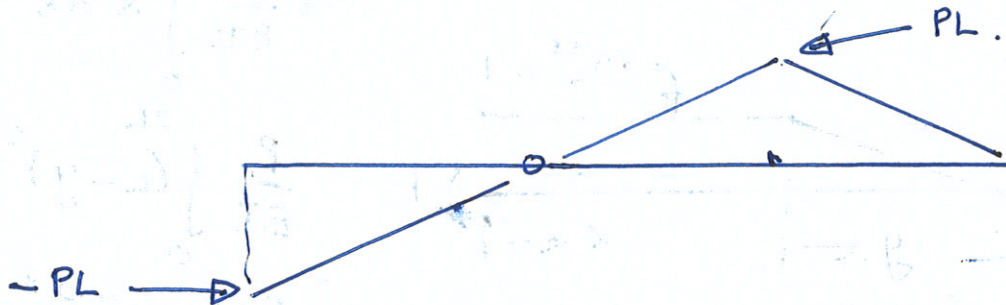


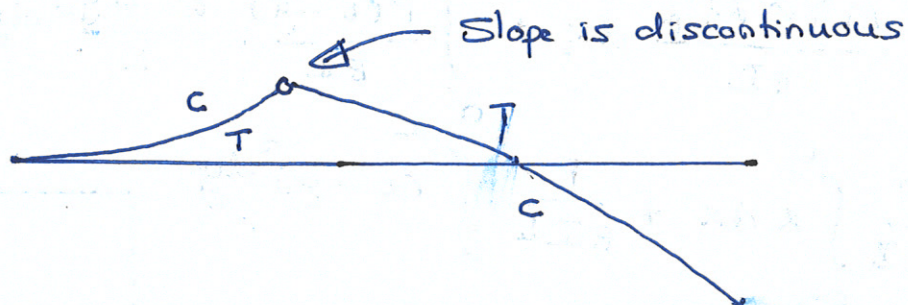
Figure 3: Elevation view of articulated cantilever beam structure.

At Point A, the cantilever is fully fixed (no movement) to a wall. Point B is a hinge. Both members have cross section properties EI . A single point load P (N) is applied at node D as shown in the figure.

Part [3a]. (2 pts). Draw and label the bending moment diagram for this problem.

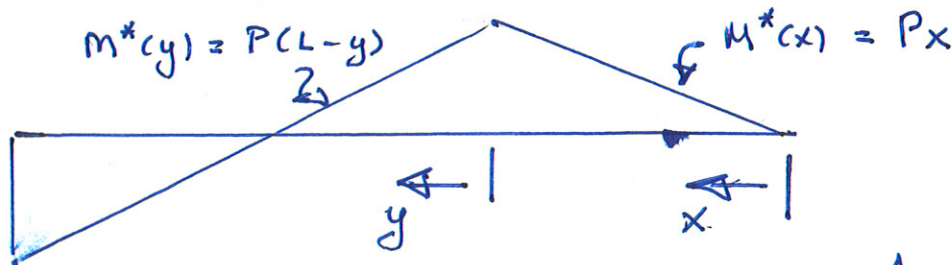


Part [3b]. (2 pts). Qualitatively sketch the deflected shape. Indicate regions of tension/compression, and any points where slope of the beam is discontinuous.

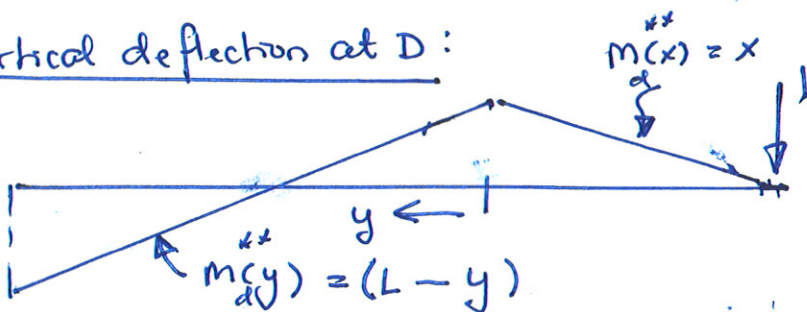


Part [3c]. (6 pts). Use the method of **virtual forces** to compute the **vertical displacement** and **end rotation** of the beam at D. Show all of your working.

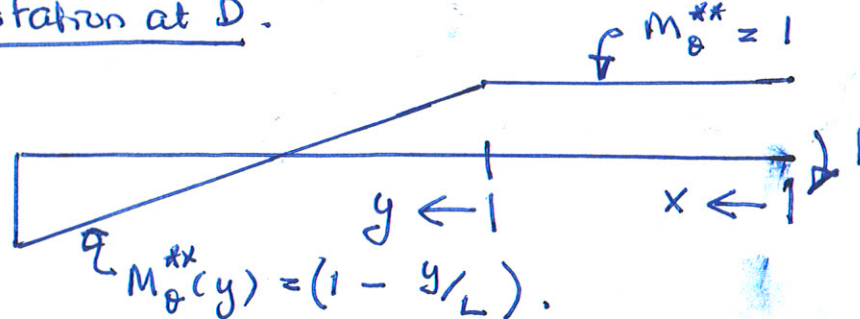
BMD due to load at D:



Vertical deflection at D:



Rotation at D:



$$\begin{aligned}\Delta_D &= \int_0^L \frac{M^*(x) m_D^*(x)}{EI} dx \\ &+ \int_0^{2L} \frac{M^*(y) m_D^*(y)}{EI} dy \\ &= \frac{P}{EI} \int_0^L x^2 dx + \frac{P}{EI} \int_0^{2L} (L-y)^2 dy \\ &= \frac{PL^3}{3EI} + \frac{2}{3} \frac{PL^3}{EI} = \frac{PL^3}{EI}\end{aligned}$$

$$\begin{aligned}\theta_D &= \int_0^L \frac{M^*(x) m_\theta^*(x)}{EI} dx + \int_0^{2L} \frac{P(L-y)}{EI} \cdot \frac{(L-y)}{L} dy \\ &= \frac{P}{EI} \int_0^L x dx + \frac{P}{EIL} \int_0^{2L} (L-y)^2 dy \\ &= \frac{PL^2}{2EI} + \frac{2PL^2}{3EI}\end{aligned}$$

$$\theta_D = \frac{7}{6} \frac{PL^2}{EI}$$

$$\boxed{\Delta_D = \frac{PL^3}{EI}, \theta_D = \frac{7}{6} \frac{PL^2}{EI}}$$

Question 4: 10 points

Problem Statement. Figure 4 is a front elevation view of a simple truss that supports vertical loads at nodes C and D.

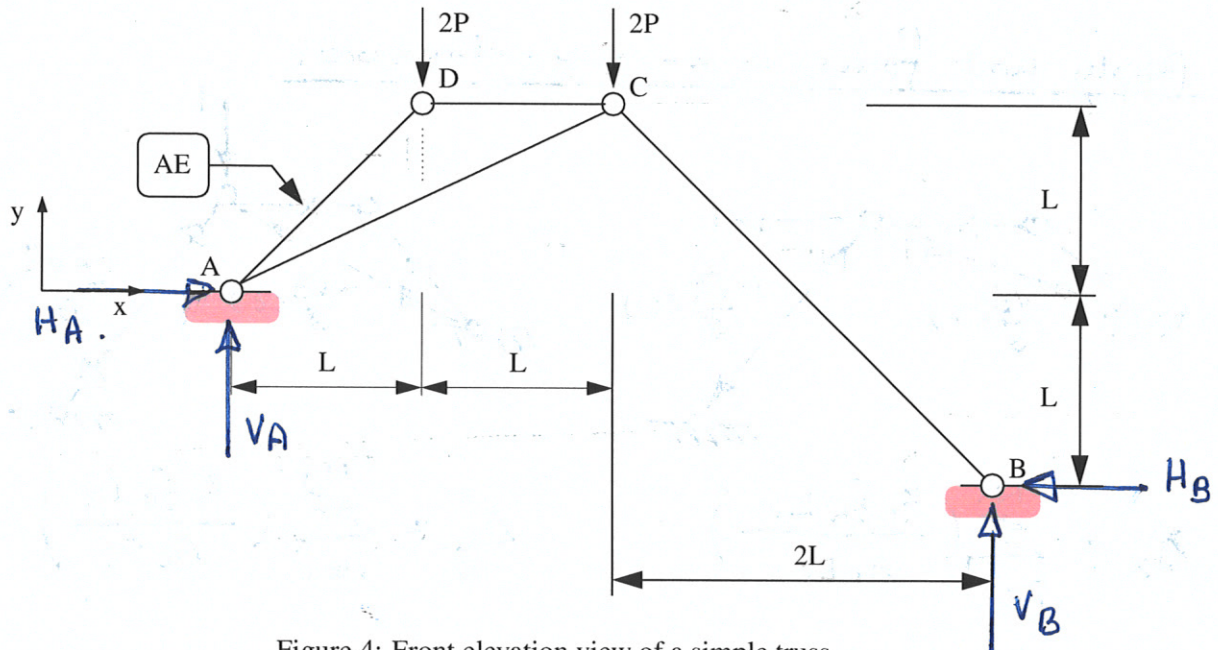


Figure 4: Front elevation view of a simple truss.

All of the truss members have cross section properties AE.

Part [4a]. (5 pts). Compute the support reactions and distribution of forces throughout the structure.

$$\sum M_A = 0 \quad -2PL - 4PL - H_B L + 4L V_B = 0$$

$$\Rightarrow 4V_B = H_B + 6P.$$

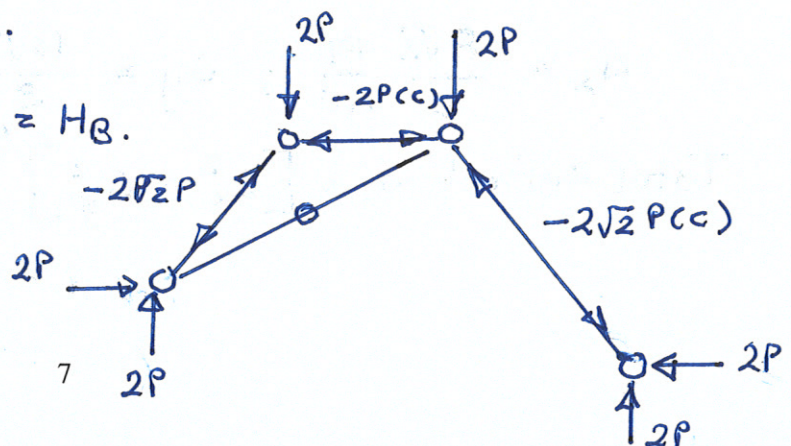
$$\sum F_x = 0 \quad H_A = H_B$$

$$\sum F_y = 0 \quad V_A + V_B = 4P.$$

Equilibrium at node B; $V_B = H_B$.

$$\Rightarrow V_A = V_B = 2P$$

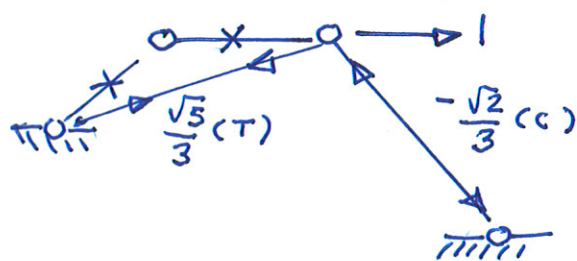
$$H_A = H_B = 2P.$$



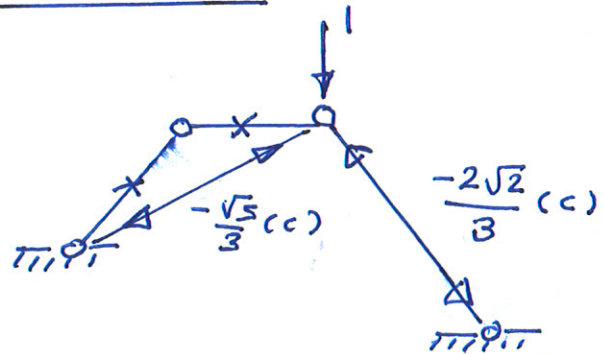
Part [4b]. (5 pts). Use the method of **virtual forces** to show that the total deflection at node C is:

$$\Delta = \frac{PL}{AE} \left[\frac{8\sqrt{10}}{3} \right]. \quad (1)$$

Apply unit forces in x- & y- directions:



$$\Delta_x = \sum_{i=1}^N \left(\frac{F_i f_h L_i}{A_i E_i} \right)$$



$$\Delta_y = \sum_{i=1}^N \left(\frac{F_i f_v L_i}{A_i E_i} \right)$$

Tabulate displacements:

Member	L/AE	F	f_h	f_v	$\frac{F f_h L}{AE}$	$\frac{F f_v L}{AE}$
AC	$\sqrt{5}L/AE$	0	$\sqrt{5}/2$	$-\sqrt{5}/3$	0	0
AD	$\sqrt{2}L/AE$	$-2\sqrt{2}P$	0	0	0	0
BC	$2\sqrt{2}L/AE$	$-2\sqrt{2}P$	$-\sqrt{2}/3$	$-\frac{2\sqrt{2}}{3}$	$\frac{8\sqrt{2}}{3} \frac{PL}{AE}$	$\frac{16\sqrt{2}}{3} \frac{PL}{AE}$
CD	L/AE	$-2P$	0	0	0	0

$$\Delta_x = \frac{8\sqrt{2}}{3} \frac{PL}{AE} ; \Delta_y = \frac{16\sqrt{2}}{3} \cdot \frac{PL}{AE}.$$

$$\text{Total deflection} = \left[\Delta_x^2 + \Delta_y^2 \right]^{1/2} = \frac{8\sqrt{10}}{3} \frac{PL}{AE}.$$

Question 5: 20 points

Problem Statement. The T-shaped beam structure shown in Figure 5 has flexural stiffness EI throughout.

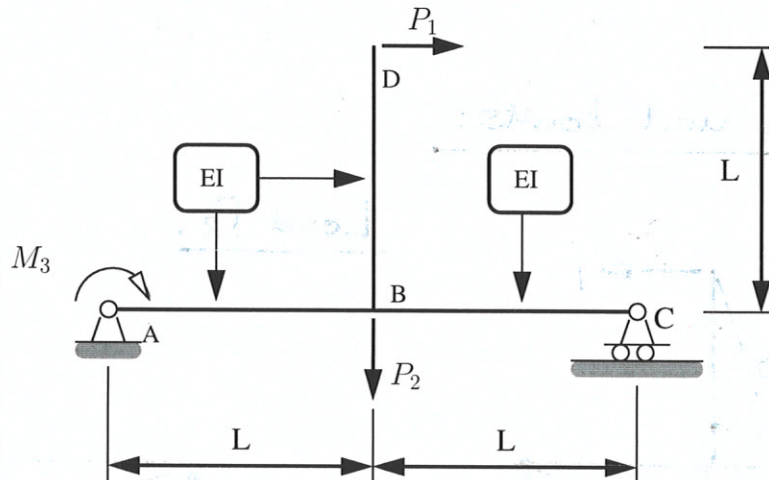


Figure 5: Front elevation view of a T-shaped beam.

Part [5a]. (5 pts). Use the **method of virtual displacements** to compute the vertical reaction force at node A.

$$\sum EWD = 0$$

$$V_A \Delta^{**} + M_3 \theta + \frac{P_1 \Delta^{**}}{2} - \frac{P_2 \Delta^{**}}{2} = 0$$

from geometry: $\theta = \frac{\Delta^{**}}{2L}$

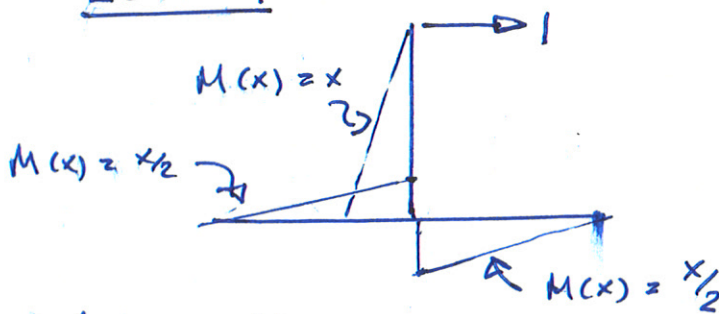
$$\Rightarrow V_A = \frac{P_2}{2} - \frac{P_1}{2} - \frac{M_3}{2L} \quad \text{--- (A)}$$

Part [5b]. (15 pts). Use the **method of virtual forces** to compute the flexibility matrix:

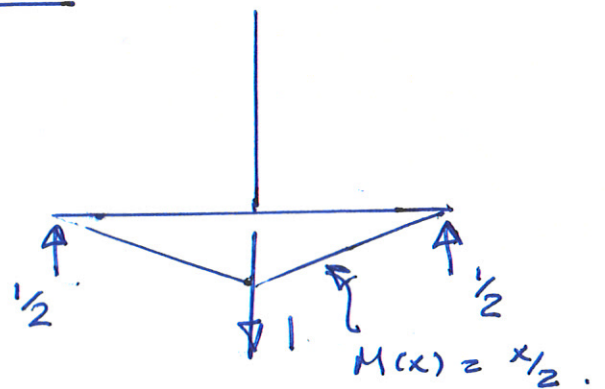
$$\begin{bmatrix} \Delta_{dx} \\ \Delta_{by} \\ \theta_A \end{bmatrix} = \begin{bmatrix} f_{11} & f_{12} & f_{13} \\ f_{21} & f_{22} & f_{23} \\ f_{31} & f_{32} & f_{33} \end{bmatrix} \begin{bmatrix} P_1 \\ P_2 \\ M_3 \end{bmatrix}. \quad (2)$$

① BMD due to unit loads:

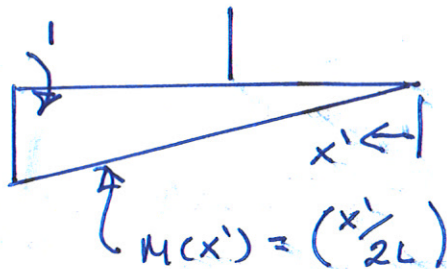
Load P_1 .



Load P_2 .



Moment M_3 :



② Flexibility Coefficients:

$$f_{11} = 2 \int_0^L \left(\frac{x}{2}\right)^2 \frac{1}{EI} \cdot dx + \int_0^L \frac{x^2}{EI} dx = \frac{L^3}{2EI}.$$

From symmetry: $f_{12} = f_{21} = 0$.

$$\begin{aligned} f_{31} &= \int_0^L -\frac{x}{2} \left(1 - \frac{x}{2L}\right) \frac{1}{EI} dx + \int_0^L \left(\frac{x'}{2}\right) \left(\frac{x'}{2L}\right) \frac{1}{EI} dx' \\ &= \frac{-L^2}{12EI}. \end{aligned}$$

Part [5b]. continued: ...

$$f_{22} = 2 \int_0^L \left(\frac{x}{2}\right)^2 \frac{1}{EI} dx = \frac{L^3}{6EI}.$$

$$\begin{aligned} f_{32} &= \int_0^L \left(\frac{x}{2}\right) \left(1 - \frac{x}{2L}\right) \frac{1}{EI} dx + \int_0^L \frac{x'}{2} \cdot \frac{x'}{2L} \cdot \frac{1}{EI} dx' \\ &= \frac{L^2}{4EI}, \end{aligned}$$

$$f_{33} = \int_0^{2L} \left(\frac{x}{2L}\right)^2 \cdot \frac{1}{EI} dx = \frac{2}{3} \frac{L}{EI}.$$

Flexibility Matrix

$$f = \begin{bmatrix} \frac{L^3}{2EI} & 0 & -\frac{L^2}{12EI} \\ 0 & \frac{L^3}{6EI} & \frac{L^2}{4EI} \\ -\frac{L^2}{12EI} & \frac{L^2}{4EI} & \frac{2}{3} \cdot \frac{L}{EI} \end{bmatrix}$$

Question 6: 10 points

Problem Statement. Consider the supported cantilevered beam structure shown in Figure 6.

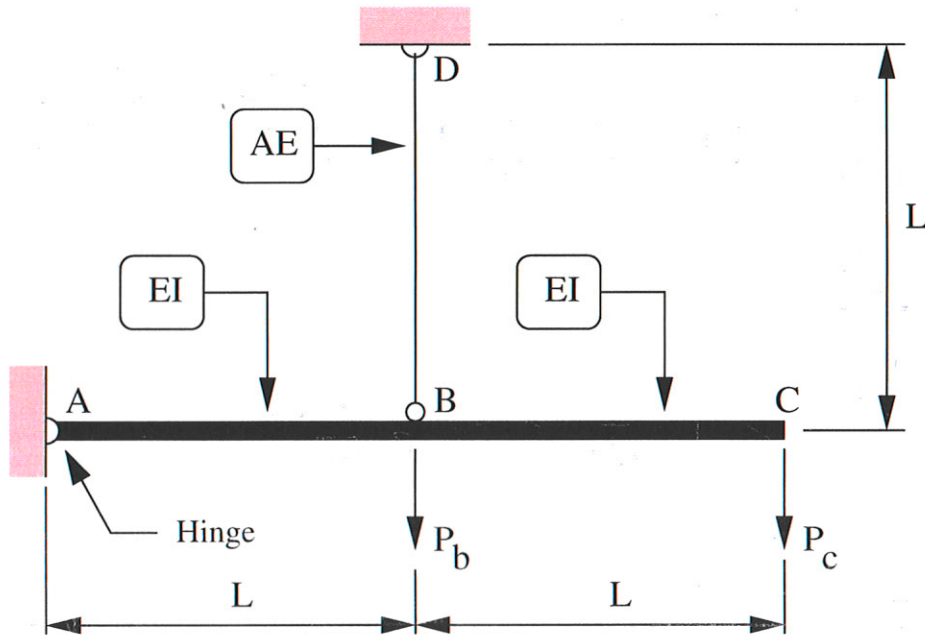
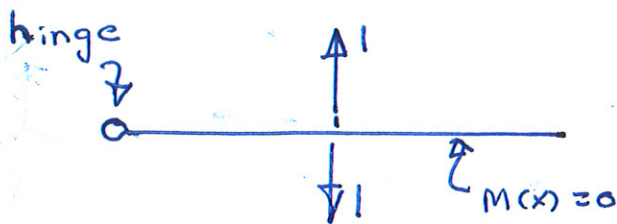


Figure 6: Front elevation view of a supported cantilevered beam structure.

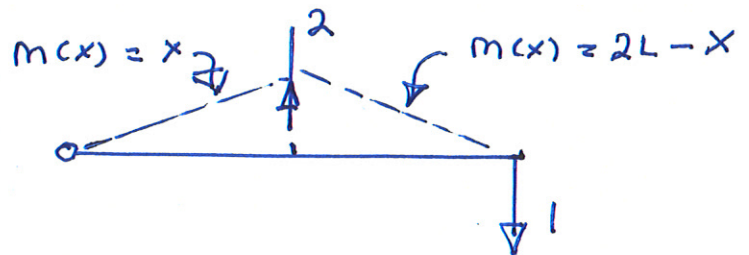
Use the principle of **virtual forces** to compute the two-by-two flexibility matrix connecting displacements at points B and C to applied loads P_b and P_c , i.e.,

$$\begin{bmatrix} \Delta_b \\ \Delta_c \end{bmatrix} = \begin{bmatrix} f_{11} & f_{12} \\ f_{21} & f_{22} \end{bmatrix} \begin{bmatrix} P_b \\ P_c \end{bmatrix}. \quad (3)$$

Apply unit force at B.



Apply unit force at C.



Question 6. continued: ...

$$f_{11} = \underbrace{\int_0^{2L} \frac{M_1(x)^2}{EI} dx}_{I_1} + \underbrace{\int_0^L \frac{f_1(x)^2}{AE} dx}_{I_2} = \left[\frac{L}{AE} \right]$$

$$f_{22} = \underbrace{\int_0^{2L} \frac{M_2(x)^2}{EI} dx}_{I_1} + \underbrace{\int_0^L \frac{f_2(x)^2}{AE} dx}_{I_2}$$

$$\left. \begin{aligned} I_1 &= 2 \int_0^L \frac{x^2}{EI} dx = \frac{2}{3} \frac{L^3}{EI} \\ I_2 &= \int_0^L \frac{2 \cdot 2}{AE} dx = \frac{4L}{AE} \end{aligned} \right\} f_{22} = \frac{2}{3} \frac{L^3}{EI} + \frac{4L}{AE}$$

$$f_{12} = \underbrace{\int_0^{2L} \frac{M_1(x) M_2(x)}{EI} dx}_{\rightarrow 0!} + \int_0^L \frac{f_1(x) f_2(x)}{AE} dx = \frac{2L}{AE}$$

Flexibility Matrix:

$$\begin{bmatrix} \Delta_b \\ \Delta_c \end{bmatrix} = \begin{bmatrix} L/AE & 2L/AE \\ 2L/AE & \frac{2}{3} \frac{L^3}{EI} + \frac{4L}{AE} \end{bmatrix} \begin{bmatrix} P_b \\ P_c \end{bmatrix}$$

