

Analysis of Beam Structures

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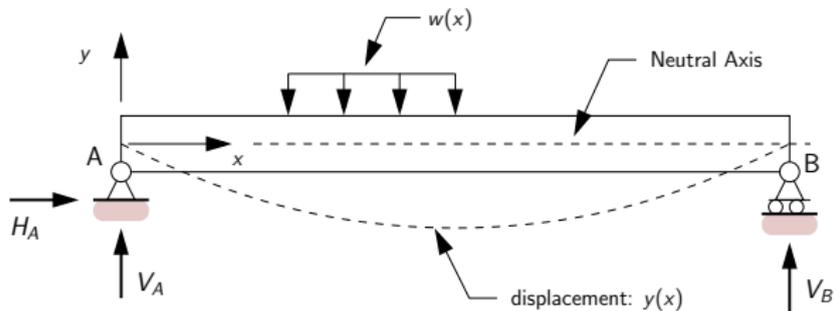
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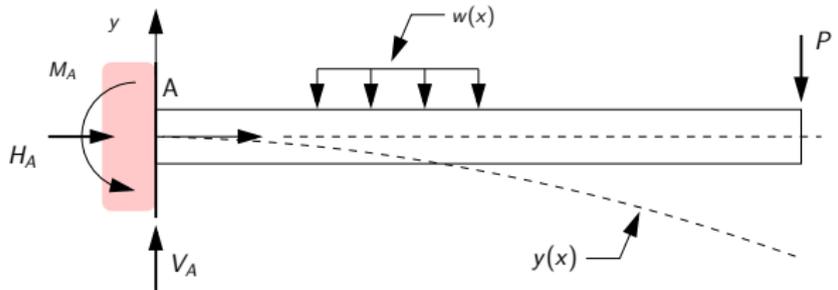
Types of Beam Structure

Types of Beam Structures

Simply Supported Beam:

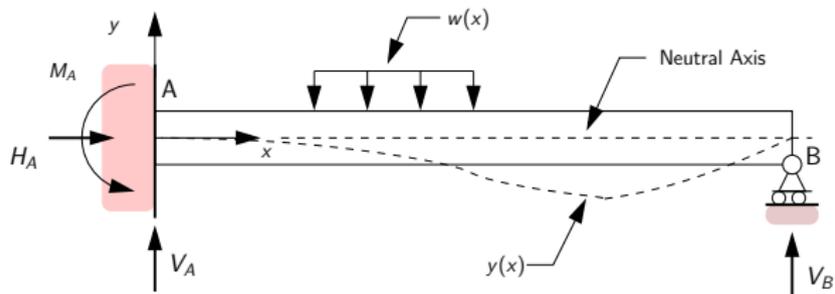


Cantilever:

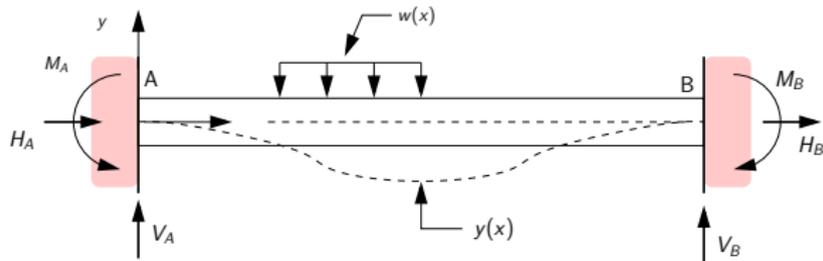


Types of Beam Structures

Supported Cantilever:



Fixed-Fixed Beam Structure:



Types of Beam Structures

Boundary Conditions

Simply Supported Beam

- $y(0) = y(L) = 0$.

Cantilever Beam

- $y(0) = 0, \frac{dy}{dx}|_{x=0} = 0$

Supported Cantilever Beam

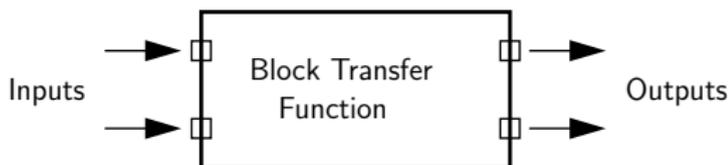
- $y(0) = y(L) = 0, \frac{dy}{dx}|_{x=0} = 0$

Fixed-Fixed Beam

- $y(0) = y(L) = 0, \frac{dy}{dx}|_{x=0} = \frac{dy}{dx}|_{x=L} = 0$

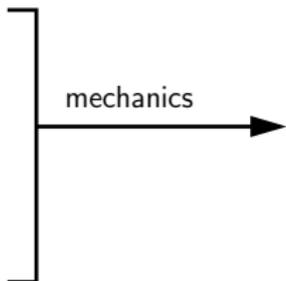
Basic Questions

Q1. What is the relationship between inputs and outputs?



Inputs

Applied loads (P and w)
Boundary conditions
Beam geometry (L and I)
Material Properties (E)



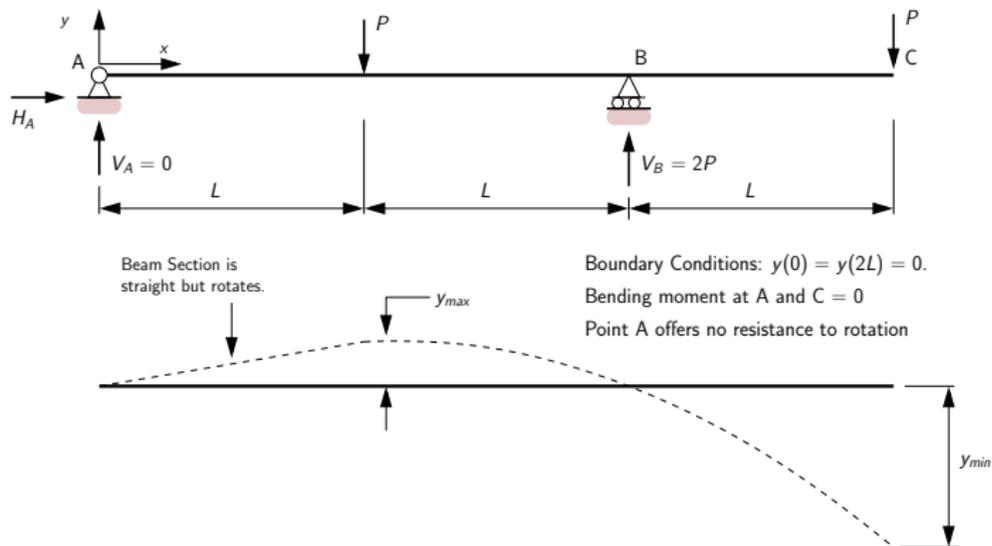
Outputs

Shear force – $V(x)$
Bending moment – $M(x)$
Axial force – $N(x)$
Displacement – $y(x)$
Rotation – $\theta(x) = \left[\frac{dy}{dx} \right]$

Decisions will be based on estimates of outputs.

Basic Questions

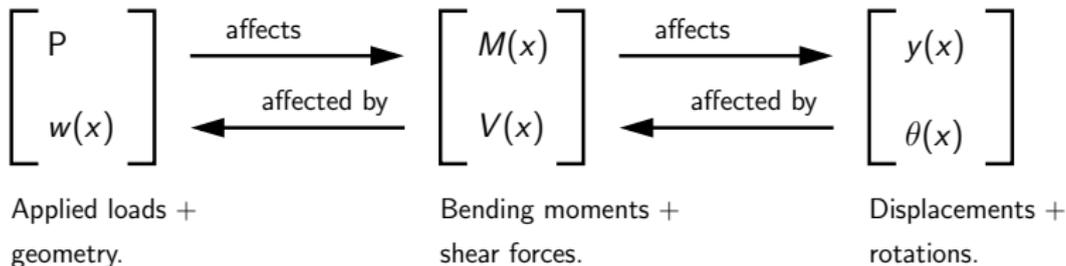
Typical problem: Given input parameters, compute $y(x)$, find location and magnitude of y_{min} and y_{max} .



For simple problems, can rely on intuition. Otherwise, need **math** and **mechanics**.

Basic Questions

Q2. What is the relationship among the outputs? Are they dependent?



We will need to work with a chain of dependencies.

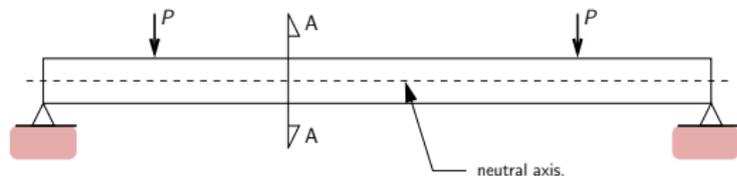
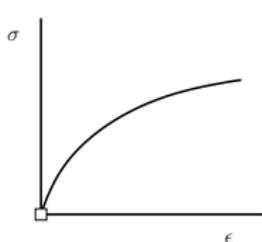
Q3. What is the relationship between $V(x)$ and $M(x)$? Are they independent? No!

We will see: $V(x) = \frac{dM(x)}{dx}$, but not always true!

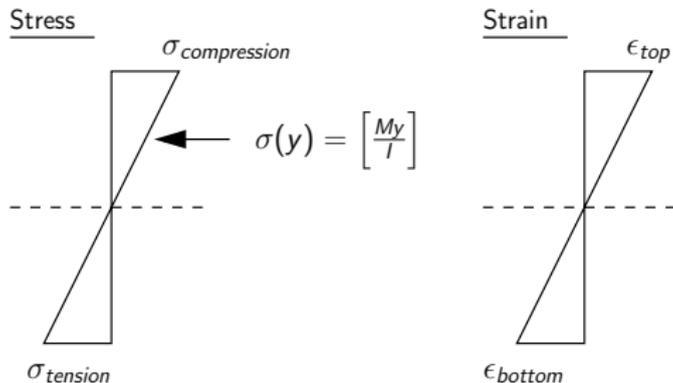
Connection to Mechanics

Connection to Mechanics

Problem Setup



Stress-Strain Relationships



Connection to Mechanics

For design purposes we need to make sure:

$$\sigma_{tension} < \sigma_{\max \text{ tension}} \quad (1)$$

and

$$\sigma_{compression} < \sigma_{\max \text{ compression}} \quad (2)$$

Also,

$$\epsilon_{\max \text{ compression}} \leq \epsilon(y) \leq \epsilon_{\max \text{ tension}} \quad (3)$$

These constraints limit the amount of load that a beam can carry.

Connection to Mechanics

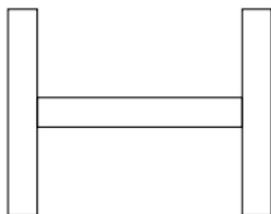
Section-Level Behavior

From a design standpoint we can reduce $\sigma(y)$ and $\epsilon(y)$ by increasing the moment of inertia in

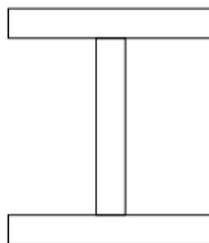
$$\sigma(y) = \left[\frac{My}{I} \right]. \quad (4)$$

To maximise I , maximize distance of material from neutral axis.

Poor Choice of Inertia



Good Choice of Inertia



Assumptions on Beam Displacements

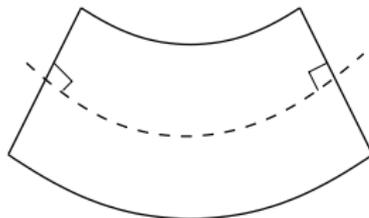
Assumptions. We will assume beam length / depth $\gg 10$.

Therefore, displacements will be dominated by flexural bending.

Undeformed Configuration



Deformed Configuration



Sections remain perpendicular to the deformed neutral axis.

This is **not the case** for **shear deformations**.